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### 2.1 List of formulae

Abbr.	Designation	Measuring unit	Formula
$\phi^{(\bullet)}$	Lead angle Self-locking during shutdown*: $2,4^{\circ} < \phi < 4,5^{\circ}$ (Self-locking out of actuation: $\phi < 2,4^{\circ}$ ) Not self-locking: $\phi > 4,5^{\circ}$	o	$\varphi$ =arctan[P <sub>h</sub> /(d <sub>2</sub> * $\pi$ )]
$\eta_{Anl}$	Lifting system efficiency rating		
$\eta_{\text{HE}}$	Worm gear screw jack efficiency rating		
а	Acceleration	m/s <sup>2</sup>	a =v / (60*t)
As	Number of load cycles		
С	Dynamic load rating	kN	
Co	Static load rating	kN	
d <sub>2</sub>	Pitch diameter	mm	
ED	Duty ratio	%/hr	ED = [lift*As/(60*v)]*100%
F <sub>dyn</sub>	Dynamic axial force (= lifting force)	kN	
F <sub>stat</sub>	Static axial force (= retention force)	kN	
HU	Lift per rotation	mm	HU= P <sub>h</sub> / i
i	Transmission ratio		
L <sub>h</sub>	Service life	h	Lh=(C/F <sub>dyn</sub> ) <sup>3</sup> *10 <sup>6</sup> /(n <sub>2</sub> *60)
n <sub>1</sub>	Input speed	min <sup>-1</sup>	
n <sub>2</sub>	Output speed	min <sup>-2</sup>	n <sub>2</sub> = n1 / i
Р	Power rating	kW	P= F <sub>dyn</sub> * v / (60*η)
Ph	Screw lead	mm	
pv-value	Surface pressure x sliding speed	N/mm <sup>2</sup> * m/min	
p <sub>zul</sub>	Permitted surface pressure	N/mm <sup>2</sup>	
t	Time	S	
T <sub>1</sub>	Drive torque	Nm	$T_1 = P^*9550 / n_1$
T <sub>2</sub>	Output torque (= screw torque)	Nm	
T <sub>A</sub>	Starting torque	Nm	T <sub>A</sub> ~ T <sub>1</sub> *1,3
V	Lifting speed	m/min	$v = n_1 * Ph / i$

 $^{(\bullet)}$  Vibrations and optimized sliding properties may affect the self-locking system. If in doubt, use a motor brake.

#### Index

Index	Configuration conforming to			
HE Worm gear screw jack	EN 1570, EN 280, EN-1756, EN 1493 (VBG 14) Lead angle:			
Anl Lifting system	$     \varphi > 4,5^{\circ} $ $     \Rightarrow $ Motor with single braking torque $     \varphi > 4,5^{\circ} $ $     \Rightarrow $ Two independent braking systems			
ки Ball screw	Standards for theatre stages and broadcasting studios BGV C1 (VBG 70)			
Tr Trapezoidal screw	Similar to configuration conforming to VBG 14, although self-locking screw is not an absolute requirement if al torque transmitting components are designed to withstand			
zulPermissible	twice the rated load.			





### 2.2 Suggestions for installation

### 2.2.1 Vertical installation



Recommended arrangement for large lifting capacities and long screws

#### Tensile load

**Design:** Configuration type 1 With climbing screw jacks (without protection tube)



Arrangement with compression-loaded screw, possible with or without additional guides

## Dimensioning of screw according to Euler's case III and critical screw speed

**Design:** Configuration type 2 With rotating screw and travelling nut



Compression-loaded screws without lateral guides

## Dimensioning of screw according to Euler's case I

**Design:** Configuration type 1 With lifting screw and protection tube



Swivelling movements require articulated mounting

## Dimensioning of screw according to Euler's case II

**Design:** Configuration type 1 With lifting screw, two guide rings and protection tube

#### S = safety clearance

See chapter 7 "Accessories" for details of all required add-on items (connecting flanges, swivel mounting bases etc.).

### 2.2 Suggestions for installation

2.2.2 Horizontal installation



Arrangement with compression- and tension-forceloaded screw With guides supplied on site

## Dimensioning of screw according to Euler's case III and critical screw speed

With screws pre-adjusted to tensile load ⇒ Dimensioning according to critical speed only

**Design:** Configuration type 2 With rotating screw and travelling nut



Intermediate bearing-support of a 12 m screw

Pict. 7

Standard once-piece screws can be made in lengths of up to 6 m (3 m for screws made of rust-free material). Longer screws are supplied in sections to make installation on site easier.



Arrangement with compression- and tension-forceloaded screw With guides supplied on site

#### Dimensioning of screw according to Euler's case II, if screw retaining force is insufficient. Otherwise, apply Euler's case III

#### Design: Configuration type 1

With clamped screw and linear traversing drive axis.



Special travelling nut

Pict. 8

If critical screw speed is exceeded (only applies to configuration type 2 = rotating screw), the screws must be adequately supported. This requires the use of our specially-supplied intermediate bearings and travelling nuts (see photos).





### 2.3 Service life L<sub>h</sub>

The design and construction of Pfaff-silberblau worm gear screw and quick-lifting jacks are based on our long years of experience in the field and they have a long service life if the instructions in the operating manuals are observed.

Screw		Gearing layout	Bearing layout	
Tr and S screw Ku ball screw		N or L	Thrust and radial bearings	
These values are for guidance only,	Calculation:	Worm-drive gearing:	<ul> <li>Calculations according to</li> </ul>	
as exact calculation is not possible	L <sub>h</sub> =(C/F <sub>dyn</sub> ) <sup>3</sup> *10 <sup>6</sup> /(n <sub>2</sub> *60)	SHE standard worm screw jack and	DIN or data supplied by	
Decisive layout factors are surface		MERKUR: approximate values according	the bearing supplier	
pressure and sliding speed		to DIN3996-D		
(pv-value, p <sub>zul</sub> )		In the case of HSE high performance worm		
Guaranteed relubrication		gear screw jack, according to DIN 3996-C		
Ideal installation		Bevel gear tooth system:		
		• SHG quick-lifting screw jacks, $L_h$ = fatigue endurable		

The maximum load values given in the catalogue data (tables of settings) are based on a service life of minimum 500 operating hours.

### 2.4 Guidelines for use

### 2.4.1 Dirt protection

- Leakproofing of all ranges by means of rotary lip seals fitted to drive shafts is standard
- Enclosed housing configuration on HSE und SHG ranges using additional seals
- Screw protection tube for type 1

#### Optional lifting screw protective covers:

- Flexible protection boots made of various materials to prevent entry of dirt and for outdoor use (water-proof)
- Spiral spring cover for use in extreme environment (flying swarf, welding sparks)

#### Special configurations:

Special configurations, e. g. for use under water or at high temperatures, are available on request

#### 2.4.2 Corrosion protection

## Corrosion protected aluminium housings fitted to the following ranges:

SHE	Size 0.5 and 1.1
MERKUR	Size M0, M1 and M2
HSE	Size 32 and 36.1
SHG	Size G25

#### Surface coating for all other sizes:

- SHE and HSE housings with series-applied SHG G25 (aluminium housing)
- MERKUR and SHG with phosphate coated housings

#### Optional corrosion protection for special configurations: Available for all ranges:

- Special coatings
- Screws and screw heads made of material 1.4305, 1.4301, 1.4571
- Worm-drive shafts made of rust-free material
- SHE range in completely rust-free material configuration

#### Corrosion protection provided by surface coatings: Available for all ranges:

- Tenifer treated (salt-bath nitrided)
- Hard-chrome plated drive shafts

### 2.4 Guidelines for use

#### 2.4.3 Items requiring regular lubrication

If access to the screw itself or the lubrication points on the housing is difficult, we recommend the use of a central lubrication system or automatic lubricant dispenser (see chapter 7.8). Our components can be supplied with the necessary screw-in connections for this purpose.



#### SHE and MERKUR range

Configuration	Lubrication point
Type 1	guide ring, protection tube, housing
	(gearing layout)
Type 2	travelling nut, housing (gearing layout)

### 2.5 Ambient temperatures

#### HSE and SHG range

Configuration	Lubrication point
Type 1	guide ring, protection tube
Type 2	travelling nut







### 2.6 Precision ratings

### 2.6.1 Axial play "a"

Axial play has no influence on positioning accuracy in the case of one-side load, as the flanks of the screw thread provide support.

Trapezoidal or buttress-thread screw	Ball screw		
Standard			
0,1 mm $\leq$ a $\leq$ 0,3 mm, depending on size	Single flange nut a $\leq$ 0,05 mm		
Modified configuration:			
Axial play can be readjusted	Initial tension depending on size of ball		
	$0,01 \text{ mm} \le a \le 0,03 \text{ mm}$		
	Pre-stressed double nut a $\leq$ 0,01 mm		



b



### 2.6.2 Lateral play "b"

#### Standard

Lateral play "b" occurs only in the case of type 1, as a result of play between the guide ring and the outer diameter of the screw. It comes up to about 0,2 mm and effects a linear calculative deviation "b" depending on stroke length. The amount of play "b" can be reduced by means of a second guide ring.

#### Special configuration

2nd guide ring with reduced play and additionally ground screw material.





The tooth profile play (when new) of 0,1 - 0,3 mm varies, depending on the size of the unit resp. centre distance of shaft to wheel, with the rate of wear.



Tooth profile play of 0,05 - 0,1 mm kept constant throughout service life of unit.

#### 2.6.4 Screw lead error

Trapezoidal screw thread conforming to DIN 103 T1; Buttress screw thread conforming to DIN 513		Ball screw thread conforming to DIN 69051T3
Whirled screw (standard) ± 0,05 mm	300 mm	Whirled screw (standard) Tolerance classification T7; P300 = 0,052 mm
Rolled screw ± 0,1 mm		Ground screw; tolerance classification T1 - 5; P300 = 0,006 - 0,023 mm
		Rolled screw; tolerance classification T9; P300 = 0,1 mm

### 2.7 Special operating areas of application







### 2.8 Dimensioning of worm gear screw jacks

### 2.8.1 Flow diagram



### 2.8 Dimensioning of worm gear screw jacks

2.8.2 Example	nit with motor	r					
Required axial fo		force F <sub>dyn</sub> 20 kN		Guides fitted on site		Yes 🖒	
Required lifting s		speed v	_ 1,9 m/mi	n		Euler's case	e III
	Required lift		_ 1200 mm	۱	Load cycles/hour	10	
	Selected config		juration See chapter		Travel per load cycle	1200 mm	
					Configuration type 1 (lif	ting screw)	
Screw		From buckling	g diagram		Tr 50x9		
Pre-selection of worm	Pre-selection of worm gear screw jack		Chapter 3.4.1		HSE 63.1		
Required drive performance		2,0 kW		Perfo	rmance limit according		
				to table	e of settings, chapt. 3.4.1	$Perr < p_{zul} = 2,3 $	<vv< td=""></vv<>
Duty ratio		11%/h		For fo	ormula, see chapt. 2.1	$ED_{vorh} < ED_{zul} = 2$	0%/h
Selected motor		2,2 kW, 150	10 min <sup>-1</sup>				
Selected			ted size H	SE 63 1	is ok		

#### 2.8.3 Manual operation for lifting elements



#### 2.8.4 Motor drive for lifting elements



#### Motor designs:

Required starting torque	T <sub>A</sub> ~ 1,3 x T <sub>N</sub>
High lifting speeds, e.g. servo drive system	Inertial mass and acceleration time are
	decisive factors for configuration





### 2.8 Dimensioning of worm gear screw jacks

#### 2.8.5 Adjustment and stopping accuracy

The level of adjustment accuracy depends mainly on the precision of the screw (see chapter 2.6). Positioning accuracy is influenced, in the case of motor-actuated units, by the electrical control and regulating system, the control system of the brake, and by the adjustment precision of the limit switches.

IMPORTANT: DO NOT allow motor-actuated components to collide with fixed end-limit stops!

### 2.9 Permitted operating settings

#### 2.9.1 General



### 2.9 Permitted operating settings

### 2.9.2 Operating factors

#### Standard worm gear screw jacks SHE and MERKUR

Reduction of ED relative to ambient temperature

Ambient temperature [°C]		50 °	60 °	70 °	80 °
in the case of screw jacks SHE and MERKUR					
Moving up poweritted duty votio	%/h	18	15	10	5
Maximum permitted duty ratio	%/10 min	36	30	20	10
CAUTION: Maximum operating temperature HE = 80 °C					

### High performance worm gear screw jack HSE

Size HSE	32	36.1	50.1	63.1	80.1
Power factor k1 [kW]	0,40	0,64	1,0	1,62	2,43
Size HSE	100.1		125.1		200.1
Power factor k1 [kW]	3,30		5,41		13,30

10"/.EC

5%.EL

0%ED

5%E

0%ED

0%.EC

60 %EC 70%ED

80%ED 100%ED

30 35 Related Time (min)

#### Power factor k<sub>1</sub>

Power factor k<sub>1</sub> is the lost energy (quantity of heat) that can be dissipated at ED 20 %/h and 20 °C ambient temperature of HSE without an outside cooling system. The steady-state temperature in this case is 80 °C.



#### Duty factor k<sub>2</sub>

The duty factor  $k_{\rm 2}$  is the correction value used to increase or decrease the permitted drive capacity  $p_{zul}$ with a deviation of 20 % ED/h. At 20 % ED/h or 30 %ED/10 min is  $k_2 = 1$ .

If the duty ratio deviates from the established settings,  $k_2$  can be determined using the diagram shown here.

#### Temperature factor k<sub>3</sub>

At normal temperatures of 20 °C, this factor is 1. In the event of fluctuating ambient temperatures (=  $\vartheta$ ), this factor is calculated as follows:





45 50 55

40



2,8 2,5 2.4

2.2

2,0

1,8

1,6

1,4

1,2

1,0

0,9

0,8

0,7

0,6

0,5

0,44 0,4

10 15 20 25

Operating Factor k<sub>2</sub>



### 2.10 Dimensioning of lifting systems

### 2.10.1 Flow diagram



#### 2.10.2 Example

#### Technical specifications:

 $\label{eq:Fges} \begin{array}{l} \mathsf{F}_{ges} = 60 \ kN \ (dyn. \ and \ stat.) \\ \mathsf{v} = 1,9 \ m/min \\ \mathsf{ED} = 20 \ \%/h \\ \mathsf{Schematic view } 4.1 \\ \mathsf{Three-phase motor} \\ \mathsf{Bevel gear box } i = 1:1 \end{array}$ 



$F_{HE} = 60 \text{ kN}/4*1.3$	Pre-selection of screw jack according to chapter 2.8	➡ HSE 63.1, Tr50x9,		
F <sub>HE</sub> = 19,5 kN	The selection of screw jack according to chapter 2.0	$\eta_{\text{HE}}$ = 0,311; $\text{P}_{\text{HE}}$ = 2,0 kW, $\eta_{\text{Anl}} \sim$ 0,8		
$P_{Anl} = 7,63 \text{ kW}$	rightarrow motor selection 7,5 kW, n <sub>1</sub> = 1500 min <sup>-1</sup>	➡ motor 132 M/4		
$T_{Anl} = 49 \text{ Nm}$	➡ T <sub>Keg</sub> = 25 Nm, i = 1:1 (chapter 4)	➡ bevel gear box K 11.13		
	$T_{GW1} = 25$ Nm, $n_1 = 1500$ min <sup>-1</sup> ; observe max. length according to $n_{krit}$ (chapter 6)	➡ connecting shaft ZR 28/38		
	$T_{GW2} = 12,5$ Nm, $n_1 = 1500$ min <sup>-1</sup> ; observe max. length according to $n_{krit}$ (chapter 6)	➡ connecting shaft ZR 24/28		

### 2.11 Schematic drive system layout



Pfaff-silberblau worm gear screw jacks and quick-lifting screw jacks can be used as either single drive units (see chapter 2.11.1) or multi-screw lifting systems (see chapter 2.11.2). Multi-screw lifting systems with mechanical synchronization are driven by a single motor, which makes them impervious to unequal load distribution and its negative effects on the synchronization of the screw jacks. Multi-screw lifting systems with electrical synchronization are distinguished by their low requirements in terms of mechanical connecting elements (and therefore lack of running noise), but they do require more sophisticated controls. Precise configuration of the drive motors, in conjunction with a master-slave layout, produces the corresponding exact synchronization of the drive systems. Once you have decided on the best layout for your needs, you can

establish which bevel gear boxes, couplings and connecting shafts are to be fitted. The fitting of pillow blocks can multiply the speeddependent length of the connecting shafts.

#### Note:

If quick-lifting screw jacks are in use, correct configuration can eliminate the need for bevel gear boxes.

### 2.11.1 Single drive unit

Worm gear screw jack - coupling - motor in configuration B3 (base-mounted)



Worm gear screw jack – coupling – IEC flange in configuration B14 or B5 (IEC flange-mounting)







### 2.11 Schematic drive system layout

### 2.11.2 Multi-screw lifting system

### 2.11.2.1 Mechanically synchronized









### 2.11.2.2 Electrically synchronized



